

Portable vibration analyzer for Equipment Diagnosis and On-site Measurements

Vibration Meter VA-12 With FFT analysis function



Compact & Lightweight

Vibration Analyzer *VA-12*

Major Application Fields

Product Development
Quality Assurance
Maintenance
Simple Diagnosis
Precision Diagnosis

Vibration measurement at various stages of product development
Pre-shipment testing, post-installation operation checks
Startup testing after periodic maintenance and servicing
Daily routine checks and monitoring of unusual vibration conditions
Measurement of problem vibrations and detection of fault sources

Vibration Meter Mode

Allows simultaneous measurement of acceleration, velocity, displacement, and acceleration crest factor



Vibration meter mode

Menu Mode

The crisp color TFT display (240 x 320 dots) is easy to read, whether outdoors, indoors, or in a dark

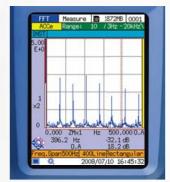
location.



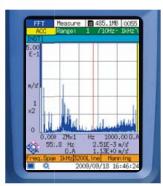
Menu

FFT Analyzer Mode

- ■Real-time analysis frequency 20 kHz
- ■Time waveform display and spectrum display with up to 3 200 spectral lines. Envelope processing also supported.
- ■Vibration waveform data recording function(10 seconds at analysis frequency 20 kHz)
 Data stored in WAVE file format on memory card (SD card).
- Timer controlled automatic measurement



Spectrum after envelope processing



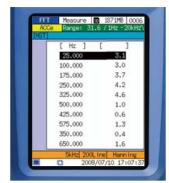
Spectrum display (3 200 lines)



Overlapping of stored data



Time waveform display



List display (top 10)

USB port allows use of unit as removable disk

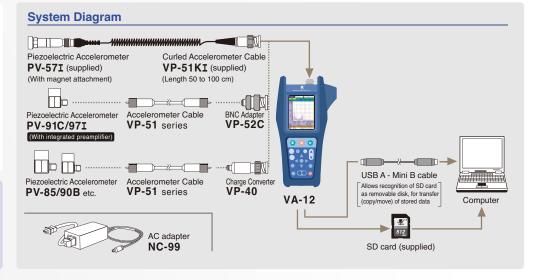


SD card slotTrigger input connectorUSB portAC adapter connector

SD cards used as memory media

Measurement data and setting data can be stored as a set on memory cards. Up to 1 000 data sets per store name are supported (max. 100 store names).





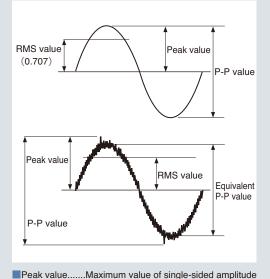
Vibration Meter Mode

Displacement / Acceleration / Velocity

Simultaneous Measurement of Three Components



Values used to express vibration magnitude



RMS value......Root mean square of instantaneous value

...Maximum difference between highest and lowest value Equivalent peak value......RMS value multiplied by $\sqrt{2}$ Equivalent P-P valueRMS value multiplied by $2\sqrt{2}$

P-P value(peak-to-peak value)

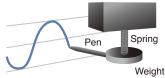
Crest factor.....Peak value/RMS value

Vibration explained

Mechanical vibrations can be represented as a complex combination of a spring and weight, as shown in the illustration on the right.

The basic physical quantities that define vibration are displacement, velocity, and acceleration.

By measuring each of these values, the vibration condition can be assessed.



Displacement explained

Unit: : µm, mm, etc.

The movement distance (travel) from a reference point is called displacement. For example, if a car travels a distance of 100 meters, the displacement value is 100 m. When considering vibrations, the movement distance of the vibrating object from the stationary rest position is the displacement, which changes between positive and negative values.



Velocity explained

Unit: mm/s, m/s, etc.

This quantity expresses the amount of change per unit of time. It is related to the vibration energy.

For example, if a car travels a distance of 100 meters in 10 seconds, the velocity is the distance (100 m) divided by the time (10 s), i.e. 10 m/s. When considering vibrations, the displacement magnitude and direction change over a short span of time, and the velocity therefore is not usually constant. The following relationship exists:

Velocity = displacement x 2 π x vibration frequency



Acceleration explained

Unit: m/s², mm/s², etc.

Acceleration is the change in velocity per unit of time.

It is proportional to the impact force or other external force.

For example, if a car traveling at a velocity of 10 m/s changes to a velocity of 30 m/s over a period of 2 seconds, the acceleration is the change in velocity (20 m/s) divided by the time (2 s), i.e. 10 m/s². When considering vibrations, the velocity and direction change over a short span of time, and the acceleration therefore is not usually constant.

The following relationship exists:

Acceleration = velocity x 2 π x vibration frequency



Usage of displacement, velocity, and acceleration

Displacement

- Measurement of vibrations in a low frequency range (below 200 Hz)
- Cases where displacement as such is critical
- Assessment of wear and damage related to static deformation, such as the effects of tensile force or compression
- Assessment of contact risks and machining precision

Velocity

- Measurement of vibrations in a medium frequency range(10 Hz to 1 kHz)
- Detection of imbalance, misalignment, bolt loosening, rattle and play etc.
- Assessment of vibration severity (ISO 10816, JIS B 0906)
- Assessment of metal fatigue

Acceleration

- Measurement of vibrations in a high frequency range (above 1 kHz)
- Detection of bearing and gear defects etc.

Vibration Meter Mode Applications

Simple Diagnosis

■ Vibration magnitude

Measuring the magnitude of vibrations is a useful diagnostic technique for ascertaining that machinery is operating normally and checking for signs of possible problems.

For example, when vibrations exceeding the reference value in the velocity range (up to 1000 Hz) are detected, the presence of an imbalance, misalignment, or loosening condition can be suspected, whereas vibrations in the acceleration range (1 kHz to about 12 to 15 kHz) point to possible bearing or gear problems.



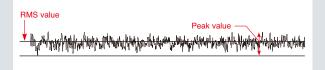
The crest factor (C.F.) is an indication of the impact characteristics of a waveform. It is determined by the ratio between the RMS and peak values. Higher crest factor values indicate a stronger impact quality.

The crest factor of acceleration measurements is useful for detecting the early stages of bearing damage.

The vibration waveform of a bearing with a fault in the initial stage is shown in the example below. Compared to the waveform of a normal bearing, the crest factor is higher.

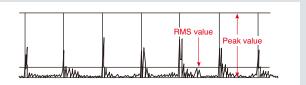
Normal bearing

(Peak value / RMS value = crest factor is small)



Bearing with spot damage

(Peak value / RMS value = crest factor is large)



Maintenance Management of Machine Equipment

By periodically measuring the vibration magnitude and comparing the results to a reference value, the equipment condition (normal or potential problem) can be diagnosed.

Using an absolute evaluation standard

ISO 10816 series (Evaluation of machine vibration by measurements on non-rotating parts).

According to ISO 10816-1:1995 / Amd. 1:2009, evaluation criteria for mechanical vibration over a specified range are to be decided by agreement between the supplier and the user of the machine, and boundary values for evaluation are to be determined in consideration of the measurement position and the support rigidity of the machine

Reference value

- A: Newly installed machinery will normally be within this range.
- · B: Long-term continuous operation allowed.
- C: Long-term continuous operation not allowed, but limited-term operation allowed.
- D: High risk of injury. Operation not allowed.

Using a relative evaluation standard (trend management)

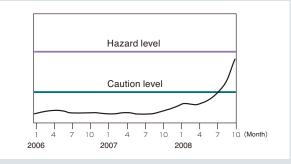
Using the normal condition as a reference, threshold values for caution and hazard conditions are set.

When the caution level is exceeded, monitoring is reinforced, and detailed diagnosis is performed when the hazard level is exceeded. A commonly used factor for setting the levels is as follows: caution level = 2 to 3 times the normal value, hazard level = 2 to 3 times the caution value.

After deciding on the vibration measurement location, measurement direction, and measurement frequency, a time series graph is commonly used for trend management, comprising measurement values and other data.

Representative zone boundary values

| Vibration velocity rms value mm/s | Range of repre | sentative zone boundary value | |
|---|---|--|---|
| 0.28 0.45 0.71 1.12 1.8 2.8 4.5 7.1 9.3 11.2 14.7 18 28 45 | Zone boundary value A/B 0.71 to 4.5 | Zone boundary value B/C 1.8 to 9.3 | Zone boundary value C/D 4.5 to 14.7 |



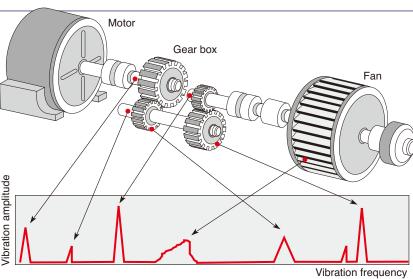
Trend management diagram

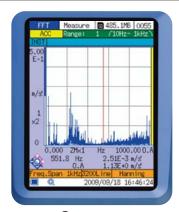
FFT Analyzer Mode

The Need for Frequency Analysis

Machinery usually comprises a variety of vibration sources such as motors, gears, bearings, fans, etc. When devising measures to minimize vibrations and when trying to locate the causes of problematic vibrations, measuring only the magnitude of vibrations often will not provide enough information. It is also necessary to perform frequency analysis, in order to determine which types of vibrations exist and what their levels are.

As shown in the illustration, the locations where vibrations occur will affect the vibration frequency. Frequency analysis makes it possible to pinpoint vibration sources with greater accuracy.

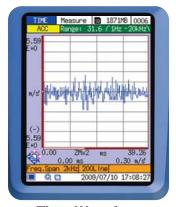




Spectrum

Vibration amplitudes are shown for each frequency. The time waveform is divided into constant intervals, and FFT analysis* is performed for these intervals. A sine wave will have only one line spectrum, but complex machine vibrations will show peaks at various frequencies.

* FFT (Fast Fourier Transform) analysis is a type of frequency analysis that is particularly suited to analyzing machine vibrations.



Time Waveform

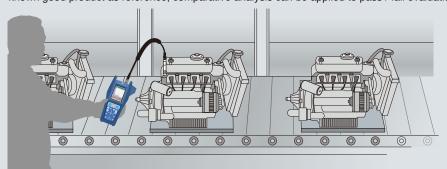
This shows the variations over time at the location of the accelerometer.

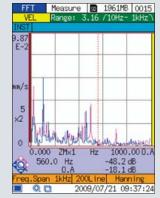
It provides information that is not available from the spectrum display, such as whether the vibration is normal or impact related, whether it has shifted upwards or downwards, etc.

FFT Analyzer Mode Applications

Product Quality Control

When testing products on manufacturing lines for unusual vibrations, frequency analysis can be very helpful. For example, when targeting a specific frequency, it can be determined whether there are vibration components in the adjacent frequency range. Using the frequency spectrum with a known good product as reference, comparative analysis can be applied to pass / fail evaluation.





Comparison to reference spectrum

(Overlapping of stored data)

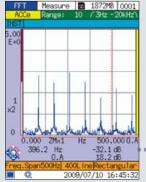
FFT Analyzer Mode Applications

Precision Diagnosis of Rotating Machinery

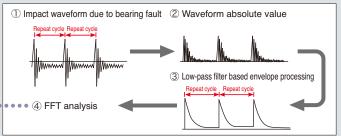
Precision diagnosis is used to determine the cause of problems as well as the extent, location etc.

Bearings

Bearing problems will cause a significant increase in acceleration values. As seen in the example, envelope analysis shows the peaks at equal intervals. When the size, number of rolling elements, axis rotation speed and other parameters are known, the primary frequency of the lined-up peaks will provide information about the problem location.



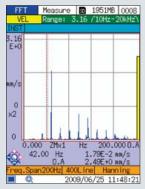
When diagnosing a bearing fault, it is necessary to know the repeat cycle of the impact waveform. This can be achieved by envelope processing, using the principle illustrated below.



Misalignment

When there is a misalignment, large vibration components that are an integral multiple of the rotation speed will appear in the axis direction.

The type of bearing joint affects the multiplication factor. In the example shown here, there are large vibration components with a factor of 3.

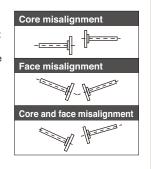


■ Misalignment explained

When two coupled rotating axes are not properly centered on relation to each other, their centers of rotation will not be in linear alignment. This is called misalignment, which can be either relative to the core or the face or a combination of the two.

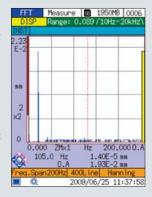
When misalignment occurs, the thrust

When misalignment occurs, the thrus load on the bearing increases due to end face runout, resulting in shorter bearing life.



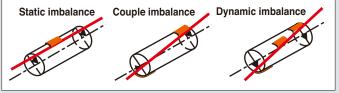
Imbalance

When there is an imbalance, large vibration components at a frequency equal to the rotation speed will appear in the circumferential direction. Vibrations of other frequencies will be largely absent. The vibration amplitude is proportional to the imbalance magnitude. At higher rotation speeds, the vibration amplitude is proportional to the square of the rotation frequency.



■ Imbalance explained

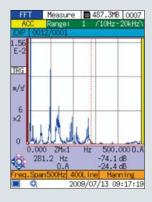
This is a condition where the center of gravity of a rotating body has shifted from the center line. There are various types of imbalance, including static imbalance, couple imbalance, and dynamic imbalance. When an imbalance occurs, the load on the bearing in the circumferential direction increases, resulting in shorter bearing life.

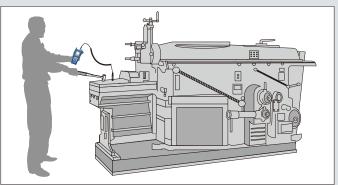


Measuring the Resonance Frequency of a Structure

When an external force at a frequency close to the resonance frequency is applied to a structure, strong vibration will occur. This can lead to breakdown of machinery, product quality degradation, and other problems. In order to guard against such risks, measuring the resonance frequency is very important. In the example shown at right, multiple resonance frequencies at 8 Hz, 98 Hz etc.

exist.





To measure the resolution frequency, the structure is struck with a hammer or similar and the resulting vibrations are subject to frequency analysis.

Specifications

| pecifications | |
|-----------------------------|---|
| tandard compliance | CE marking, Chinese RoHS (export model for China only), |
| | WEEE Directive |
| put section | |
| Number of measurement | 1 |
| channels | |
| Connector type etc. | BNC, CCLD 18 V 2 mA, (CCLD24 V 4 mA available as factory option) |
| Sensor | Piezoelectric Accelerometer PV-57I (supplied) |
| Input range | |
| | 00 to 0.999 mV/(m/s²) |
| | 10, 31.6, 100, 316, 1 000, 3 160, 10 000 m/s² (rms) |
| VEL (Velocity) | 31.6, 100, 316, 1 000, 3 160, 10 000, 31 600 mm/s (rms) |
| | 0.89, 2.83, 8.94, 28.3, 89.4, 283, 894 mm (EQp-p) |
| | 0 to 9.99 mV/(m/s²), using PV-57I |
| | 1, 3.16, 10, 31.6, 100, 316, 1 000 m/s² (rms) |
| | 3.16, 10, 31.6, 100, 316, 1 000, 3 160 mm/s (rms) 0.089, 0.283, 0.894, 2.83, 8.94, 28.3, 89.4 mm (EQp-p) |
| | 0 to 99.9 mV/(m/s²) |
| | |
| | 0.1, 0.316, 1, 3.16, 10, 31.6, 100 m/s ² (rms) 0.316, 1, 3.16, 10, 31.6, 100, 316 mm/s (rms) |
| | 0.0089, 0.0283, 0.0894, 0.283, 0.894, 2.83, 8.94 mm (EQp-p) |
| | (using PV-57I, High-pass filter 3 Hz, Low-pass filter 20 kHz) |
| | 0.02 to 141.4 m/s ² (rms) Continuous measurement, 1 Hz to 5 kHz |
| Instantaneous | 700 m/s ² |
| maximum acceleration | |
| VEL (Velocity) | 0.2 to 141.4 mm/s (rms) at 159.15 Hz |
| DISP (Displacement) | |
| | ncy range (electrical characteristics) |
| ACC (Acceleration) | r |
| VEL (Velocity) | 3 Hz to 3 kHz |
| DISP (Displacement) | 3 Hz to 500 Hz |
| Acceleration envelope curve | 1 kHz to 20 kHz |
| Filters | |
| Prefilters | |
| | 1 Hz (acceleration only), 3 Hz, 10 Hz, 1 kHz (-10 % point), cutoff slope -18 dB/oc |
| Low-pass filter | |
| Acceleration enve | |
| High-pass filter | 1 kHz (-10 % point), cutoff slope -18 dB/oct |
| Inherent noise | High-pass filter 3 Hz, Low-pass filter 20 kHz, lowest range setting |
| ACC (Acceleration) | 0.01 m/s ² (rms) or less |
| VEL (Velocity) | 0.1 mm/s (rms) or less |
| DISP (Displacement) | 0.01 mm (EQp-p) or less |
| A/D conversion | 24 bit ΔΣprinciple, 51.2 kHz |
| Dynamic range | Maximum 110 dB (Acceleration) |
| bration meter mode | |
| ACC (Acceleration) | m/s ² rms value, waveform peak value, crest factor |
| VEL (Velocity) | mm/s rms value |
| DISP (Displacement) | mm EQp-p |
| T mode | Time waveform, spectrum, Acceleration envelope curve |
| Analysis points | 512, 1 024, 2 048, 4 096, 8 192 (3 200 lines) |
| Time window functions | Rectangular, Hanning, flat-top |
| Processing | Linear average, maximum, exponential averaging, instantaneous value |
| Frequency span | 100 Hz, 200 Hz, 500 Hz, 1 kHz, 2 kHz, 5 kHz, 10 kHz, 20 kHz |
| Display | |
| Spectrum | Top 10 list, graph display (excluding DC) |
| Zoom | X axis: x1, x2, x4, x8, x16 |
| | Y axis: 2^N , $N = 0$ to 10 (x1 to x1024) |
| | ith stored data in spectrum mode |
| Time wave form | Graph display |
| Zoom | X axis : x1, x2, x4, x8, x16, x32 |
| | Y axis: 2^N , N = 0 to 14 (x1 to x16 384) |
| igger | |
| Trigger source | |
| External signal | Triggered at falling edge of signal at external trigger input |
| Input level | Triggered when time waveform crosses a preset level |
| | Trigger level can be set in steps of 1/8 of full scale on one-sided amplitude |
| Slope | +/- trigger operation |
| Trigger operation | |
| Free-run | Processing always carried out, regardless of trigger condition |
| | I I because in a coming of the because this acritical control |
| Repeat | Processing carried out whenever triggering occurs |
| Single | Processing carried out once only when triggering occurs |
| | |

| Pretrigger | | Processing starts from data 1/8 frame time ahead | |
|-----------------------------|------------------------|--|--|
| Display | | Color TFT LCD, 240 x 320 dots, with backlight | |
| | | Japanese display, English display, Time display | |
| Warning indication | | LED (lights up in red to indicate overload) | |
| M | emory | | |
| | Memory media | SD cards (max. 2 GB)* | |
| | Store files | Sets of measurement values and parameters can be stored on memory car | |
| | | 1 000 data saved as one store name. Max. number of store names: 100 | |
| | Parameter setting | Up to 5 parameter sets can be stored in unit | |
| memory | | Parameter settings can be stored on memory card | |
| | Wave files | Up to 10 seconds per file (frequency range 20 kHz) | |
| | | Vibration waveform recorded during FFT processing | |
| | | available when using a computer. | |
| | BMP files | Screen capture can be saved as BMP files. | |
| | Recall function | Measurement data can be read from memory card and redisplayed on screen. | |
| | Resume function | Settings are memorized when power is turned off and can be restored at next power-or | |
| Inp | out/output section | | |
| Trigger input connector | | TTL level, BNC-mini plug, 2.5 mm dia. (for CC-24) | |
| | USB port Removable | Allows use of memory card inserted in unit as removable storage | |
| | disk function | device (removable storage device class) | |
| Po | ower | | |
| | DC12 V (11 to 15 V) | AC adapter NC-99, eight IEC R6 (size AA) batteries | |
| | | (23°C, normal operation, backlight off) | |
| Battery life | | Approx. 12 hours | |
| Current consumption | | 145 mA (normal operation, backlight off) | |
| Power consumption | | Approx. 10 VA (in case of AC 100 V (NC-99)) | |
| (primary side) | | | |
| An | nbient temperature and | | |
| humidity conditions for use | | | |
| | Accelerometer | -20 °C to +70 °C, 90 % RH or less | |
| | Main unit | -10 °C to +50 °C, 90 % RH or less (no condensation) | |
| Dimensions, Weight | | 213 (H) x 105 (W) x 36 (D) mm; Mass Approx. 850 g (incl. | |
| | | batteries, with protective cover, PV-57I connected) | |
| Supplied accessories | | Piezoelectric Accelerometer PV-57I, Curled cable, Magnet attachment | |
| | | IEC R6 (size AA) battery x 8, SD card, Protective cover, Shoulder belt | |
| | | 1 | |

Option

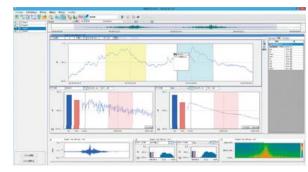
| Option | | | | |
|-----------------------------|----------|--|--|--|
| Name | Model | | | |
| Waveform Analysis Software | AS-70 | | | |
| Piezoelectric accelerometer | Various | | | |
| BNC Adapter | VP-52C | | | |
| Charge converter | VP-40 | | | |
| SD-CARD 512 MB* | MC-51SS1 | | | |
| SD-CARD 2 GB* | MC-20SS2 | | | |
| BNC-mini plug Cable | CC-24 | | | |
| AC Adapter | NC-99 | | | |
| AC Adapter | NC-99 | | | |

*Use only RION supplied cards for assured operation

Option

Waveform Analysis Software As-70

AS-70 allows post-processing using stored waveform file data from VA-12





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